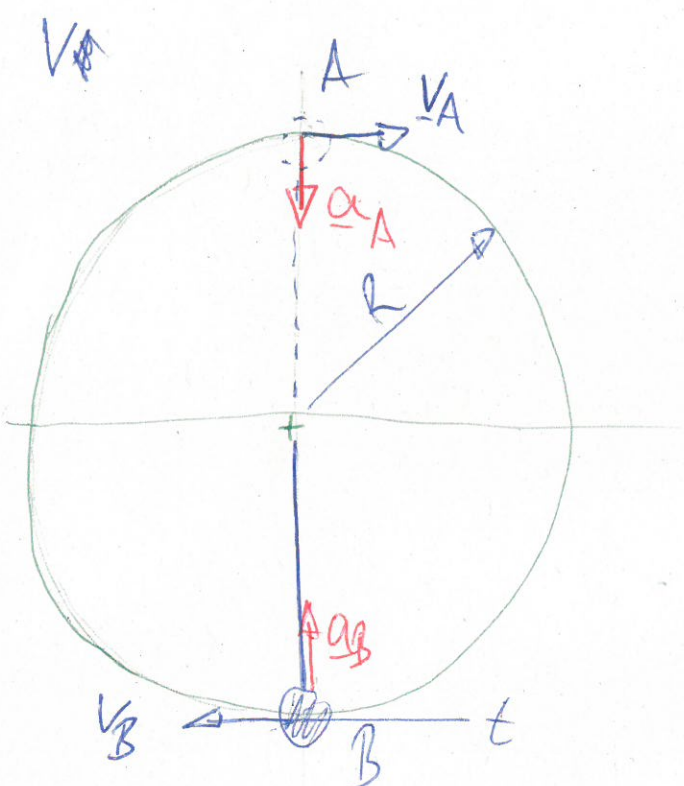
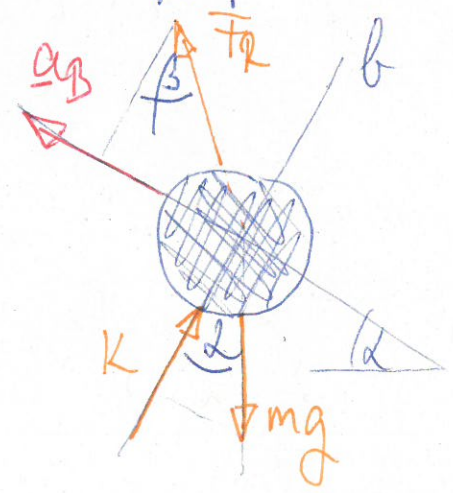


$m, l, \alpha, \beta, (\mu = 0, \text{ síma felület})$
 1) $V_A = ?$, hogy az "m" tömeg
 ne valjon el a kégyztes tól a B pont-
 ban?
 2) Kégyztes erő az A pontban?
 $\mathcal{U} = 0$



1) \mathcal{U} az a B pontban



éppen nem valjon el $K = 0!$
 $m \underline{a} = \underline{F}$
 $t: m a_{Bt} = 0 \rightarrow a_{Bt} = 0$ sebesség max
 (1) n: $m a_{Bn} = F_R \sin \beta - mg \sin \alpha$
 (2) b: $m \cdot 0 = F_R \cos \beta - mg \cos \alpha + K$
 $a_{Bn} = \frac{v_B^2}{R}, K = l \alpha \sin \beta$

V_A munkatevével számoljuk

A: t_1 , B: t_2

$$T_2 - T_1 = W_{12} = \int_{t_1}^{t_2} \underline{F} \cdot \underline{v} dt$$

$$\underline{F} = m\underline{g} + \underline{K} + \underline{F}_R$$

$\underline{K} \cdot \underline{v} = 0, \underline{F}_R \cdot \underline{v} = 0$ idealis
keijzer

$$T_2 - T_1 = W_{12} = -(U_2 - U_1)$$

$$\underline{G} = m\underline{g} = -\frac{\partial U_g}{\partial \underline{r}}, U_g = mgz$$

$$\frac{1}{2} m v_B^2 - \frac{1}{2} m v_A^2 = -(0 - 2 l \sin \beta \sin \alpha \cdot mg)$$

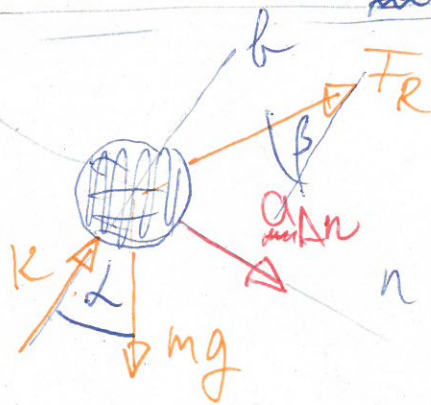
$$v_A^2 = v_B^2 - 4 g l \sin \beta \sin \alpha \quad (3)$$

$$F_R = mg \frac{\cos \alpha}{\cos \beta}, \quad a_{Bn} = g \left(\frac{\cos \alpha \sin \beta}{\cos \beta} - \sin \alpha \right)$$

$$v_B^2 = R a_{Bn}$$

$$v_A^2 = g l \sin \alpha \beta (\cos \alpha \sin \beta - \sin \alpha)$$

2) SETA' at A point



$$m \underline{a} = \underline{F}$$

t: $m a_{\underline{A}} = 0 \rightarrow v_A$ minimalis

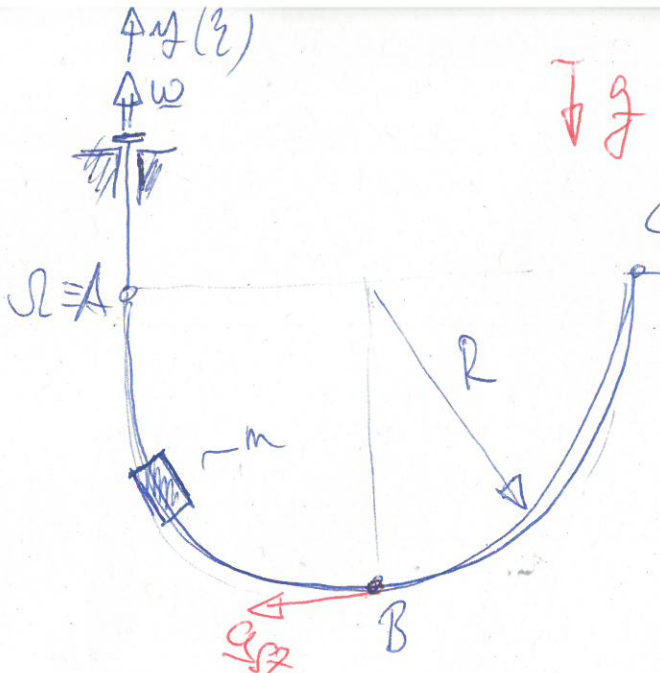
n (1) n: $m a_{An} = F_R \sin \beta + mg \sin \alpha$

(2) b: $m 0 = F_R \cos \beta - mg \cos \alpha + K$

(3) $a_{An} = \frac{v_A^2}{R}$ v_A Wert

(3) \rightarrow (1) $\rightarrow F_R = \dots mg \left(\frac{\cos \alpha}{\cos \beta} - 6 \frac{\sin \alpha}{\sin \beta} \right)$

$F_R \rightarrow$ (2) $\rightarrow K = \frac{6 \sin \alpha}{\cos \beta} mg$



$\mu=0, m=0.2 \text{ kg}, R=0.1 \text{ m}$
 $\omega=8.14 \text{ rad/s}$

m mialta A pontból, alás
 helyetben

- 1) B pontban a kinyúlás?
- 2) $v_C = ?$

$$\underline{\omega} = \begin{bmatrix} 0 \\ 0 \\ \omega \end{bmatrix}$$

alás $\{x, y, z, A\}$
 mozgás $\{\xi, \zeta, \eta, \Omega\}$ (horizontál)
 dinamika alapvetés B pontban

$$\underline{I} \underline{\ddot{\theta}} = \underline{F} \quad m \underline{a} = \underline{F}, \quad \underline{a} = \underline{\dot{v}} + \underline{a}_{sz} + \underline{a}_{cor}$$

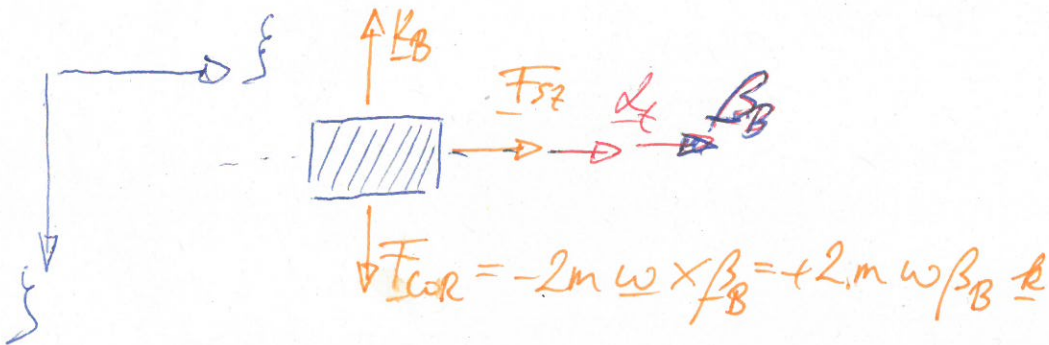
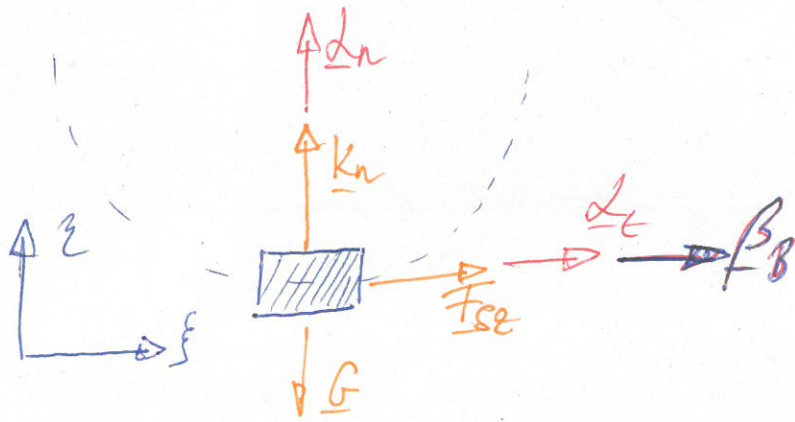
$$m \underline{\dot{v}} + m \underline{a}_{sz} + m \underline{a}_{cor} = \underline{F} \quad \rightarrow m \underline{\dot{v}} = \underline{F} + \underline{F}_{sz} + \underline{F}_{cor} = \underline{F}_{tot}$$

$$\underline{a}_{sz} = \underline{\dot{\omega}} \times \underline{r}_{CB} + \underline{\omega} \times (\underline{\omega} \times \underline{r}_{CB}) = \begin{bmatrix} -R\omega^2 \\ 0 \\ 0 \end{bmatrix}$$

$$\underline{a}_{cor} = 2 \underline{\omega} \times \underline{\beta}_B$$

$$\underline{v}_B = \underline{v}_{sz} + \underline{\beta}_B, \quad \underline{v}_{sz} = \underline{v}_A + (\underline{\omega} \times \underline{r}_{AB}) = \begin{bmatrix} 0 \\ 0 \\ -R\omega \end{bmatrix}, \quad \underline{\beta}_B = \begin{bmatrix} \beta_B \\ 0 \\ 0 \end{bmatrix}$$

β_B munkatétel segítségével! ...



mechanik (relativ)

$$\frac{1}{2} m \beta_B^2 - \frac{1}{2} m \beta_A^2 = \int_{t_1}^{t_2} F_{rel} \beta dt = \int_{t_1}^{t_2} (F_B + F_{se} \beta + F_{cor} \beta) dt$$

$F = K + G$ $K \cdot \beta = 0$, $K \perp \beta$, $G = - \frac{\partial U_g}{\partial \beta} = - \frac{mg}{\partial \beta}$

$$F = \begin{bmatrix} F \\ G \\ \xi \end{bmatrix}$$

$$\int_{t_1}^{t_2} F \cdot \beta dt = \int_{r_A}^{r_B} -dU_g = U_g(r_A) - U_g(r_B) = mgR - \textcircled{0}$$

$$\int_{t_1}^{t_2} F_{se} \cdot \beta dt = \int_{r_A}^{r_B} -dU_{se} = \frac{1}{2} m \omega^2 R^2 - 0$$

$$F_{se} = -m a_{se} = \begin{bmatrix} m \omega^2 R \\ 0 \\ 0 \end{bmatrix} \rightarrow U_{se} = -\frac{1}{2} m \omega^2 \xi^2$$

1) Coriolis erő mérése

$$F_{\text{Cor}} \beta = (2 \underline{\omega} \times \beta) \cdot \beta = 0!$$

$$\beta_A = \phi$$

$$\frac{1}{2} m \beta_B^2 + mgR + \frac{1}{2} m \omega^2 R^2 \Rightarrow \beta_B = \sqrt{2gR + \omega^2 R} = 1.528 \left[\frac{\text{m}}{\text{s}} \right]$$

dinamika algebrája

$$m \underline{a} = \underline{F} + \underline{F}_{\text{sz}} + \underline{F}_{\text{Cor}} = \underline{K} + \underline{G} + m \begin{bmatrix} R\omega^2 \\ 0 \\ 0 \end{bmatrix} + 2m\omega \beta_B \begin{bmatrix} 0 \\ 0 \\ 1 \end{bmatrix}, \quad \underline{K} = \begin{bmatrix} 0 \\ K_n \\ -K_k \end{bmatrix}$$

$$\{ m a_t = m R \omega^2 \Rightarrow a_t = R \omega^2 = 3.77 \left[\frac{\text{m}}{\text{s}^2} \right]$$

$$\{ m a_n = -mg + K_n \quad a_n = \frac{\beta_B^2}{R}$$

$$\{ 0 = -K_k + 2m\omega \beta_B \Rightarrow K_k = 3.76 \text{ [N]}, \quad K_n = 6.64 \text{ [N]}$$

2) munkatétel relatív rendszerben A & C pont között

$$\frac{1}{2} m \beta_C^2 - \frac{1}{2} m \beta_A^2 = \int_{t_A}^{t_C} \underline{F} \beta + \underline{F}_{\text{sz}} \beta + \underline{F}_{\text{Cor}} \beta dt$$

• valódi erő mérése $U_g(r_A) - U_g(r_C) = 0$ azonos pont. Mivel

$$\Rightarrow (-0 + (-\frac{1}{2} m \omega^2 (2R)^2))$$

$$\beta_C = 2R\omega = 1.228 \left[\frac{\text{m}}{\text{s}} \right]$$

$$\underline{V}_C = \beta_C + \underline{v}_{\text{sz}} = \begin{bmatrix} 0 \\ 2R\omega \\ 0 \end{bmatrix} + \begin{bmatrix} 0 \\ 0 \\ -2R\omega \end{bmatrix} = 2R\omega \begin{bmatrix} 0 \\ 1 \\ -1 \end{bmatrix}, \quad \underline{V}_C = \begin{bmatrix} 0 \\ 1.228 \\ -1.228 \end{bmatrix} \left[\frac{\text{m}}{\text{s}} \right], \quad V_C = 1.737 \left[\frac{\text{m}}{\text{s}} \right]$$